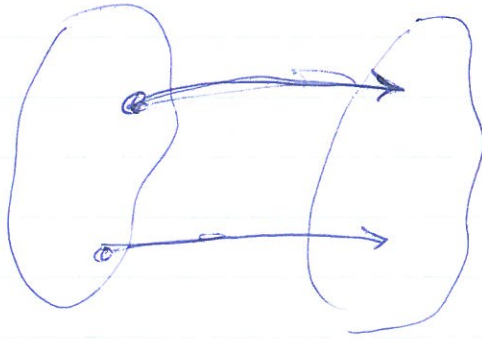


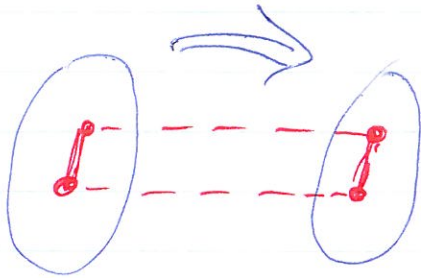
Strains - 1

Strains

In any motion : each point of body gets displacement

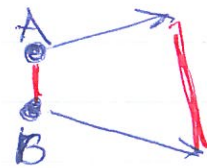


Rigid body motion : material lines do not elongate (or shorten)



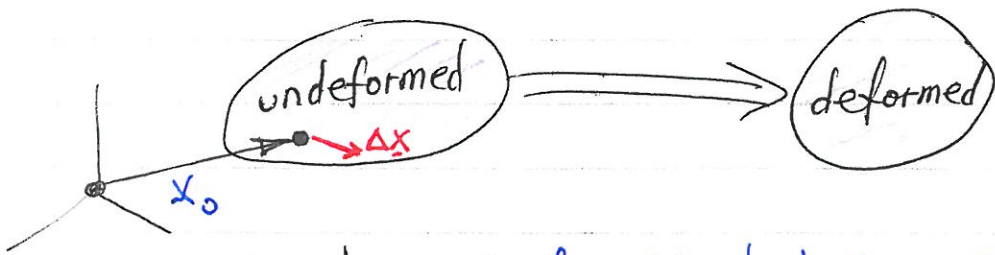
Deformation (strain) : material lines change length

because :



different
displacements
of points A, B

Non-uniformity of displacement field



Displacement of material pt \underline{x}_0 : $u(x_0)$

at neighbouring pt: $u(x_0 + \Delta x)$

If Δx is small

$$u_i(x_0 + \Delta x) \approx u_i(x_0) + \left. \frac{\partial u_i}{\partial x_j} \right|_{x_0} \Delta x_j \quad (i=1,2,3)$$

or

$$\Delta u_i = \left. \frac{\partial u_i}{\partial x_j} \right|_{x_0} \Delta x_j$$

describes non-uniformity of displacements, near $(\cdot) x_0$

↓ components of:

⇒ Displacement gradient tensor \underline{D}

displacement
gradient tensor

$$\underline{D} = \frac{\partial u_i}{\partial x_j} \underline{e}_i \underline{e}_j$$

so that

$$\Delta \underline{u} = \underline{D} \cdot \Delta \underline{x}$$

difference in displac.
of two points



Decompose into sym & antisym

$$\underline{D} = \underline{\epsilon} + \underline{\omega} = \left[\frac{1}{2} (D_{ij} + D_{ji}) + \frac{1}{2} (D_{ij} - D_{ji}) \right] \underline{e}_i \underline{e}_j$$

$$= \underbrace{\left[\frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \right]}_{\epsilon_{ij}} + \underbrace{\left[\frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right) \right]}_{\omega_{ij}} \underline{e}_i \underline{e}_j$$

$$\underline{\epsilon} - \text{strain tensor} = \epsilon_{ij} \underline{e}_i \underline{e}_j$$

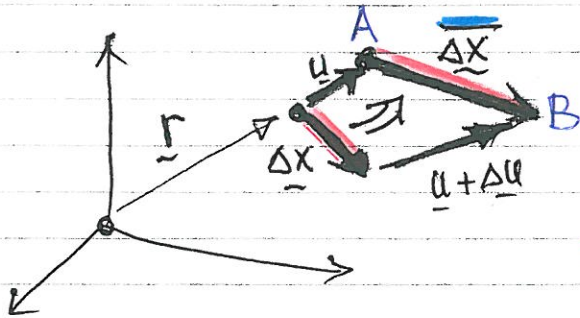
$$\underline{\omega} - \text{rotation tensor} = \omega_{ij} \underline{e}_i \underline{e}_j$$

(Have to justify these names)

Mechanical Meaning of strain components ϵ_{ij}

Consider material line $\underline{\Delta X}$ that transforms into $\underline{\bar{\Delta X}} = \underline{AB}$

$\underline{\Delta u}$: Displac. vector change (along $\underline{\Delta X}$)



position vector of point B: $\underline{r} + \underline{\Delta X} + \underline{u} + \underline{\Delta u}$
 position vector of point A: $\underline{r} + \underline{u}$

$$\underline{\bar{\Delta X}} = (\underline{r} + \underline{\Delta X} + \underline{u} + \underline{\Delta u}) - (\underline{r} + \underline{u}) = \underline{\Delta X} + \underline{\Delta u}$$

$$\Delta S^2 = (\text{length before})^2 = \underline{\Delta X} \cdot \underline{\Delta X}$$

$$\begin{aligned} \bar{\Delta S}^2 = (\text{length after})^2: \underline{\bar{\Delta X}} \cdot \underline{\bar{\Delta X}} &= (\underline{\Delta X} + \underline{\Delta u}) \cdot (\underline{\Delta X} + \underline{\Delta u}) = \\ &= \underbrace{\underline{\Delta X} \cdot \underline{\Delta X}}_{\Delta S^2} + \underbrace{2 \underline{\Delta u} \cdot \underline{\Delta X}}_{2 \Delta u_i \Delta x_i} + \underbrace{\underline{\Delta u} \cdot \underline{\Delta u}}_{\Delta u_i \Delta u_i} \end{aligned}$$

Use: $\Delta u_i = \frac{\partial u_i}{\partial x_j} \Delta x_j$

$$\Rightarrow \bar{\Delta S}^2 - \Delta S^2 = 2 \underbrace{\frac{\partial u_i}{\partial x_j}}_{D_{ij}} \Delta x_i \Delta x_j + \frac{\partial u_i}{\partial x_j} \frac{\partial u_i}{\partial x_k} \Delta x_k \Delta x_j$$

Assumption (throughout the course): small $\partial u_i / \partial x_j$

Small geometry changes

\Rightarrow second term neglected

Use $D = \epsilon + \omega$

$$= 2 \epsilon_{ij} \Delta x_i \Delta x_j + 2 \omega_{ij} \Delta x_i \Delta x_j$$

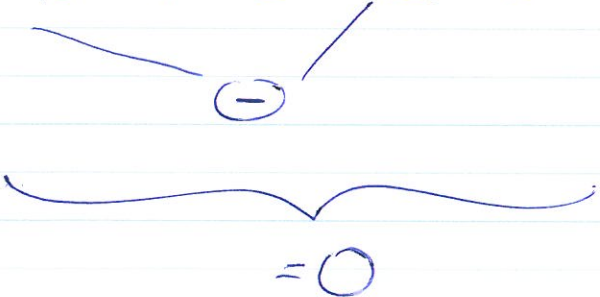
$$= 0 \text{ since } \omega_{ij} = -\omega_{ji}$$

$\Rightarrow \omega_{ij}$ does not affect length changes

Comment on ω -term:

$$\omega_{11} = \omega_{22} = \omega_{33} = 0$$

$$\omega_{ij} \Delta X_i \Delta X_j = \omega_{12} \Delta X_1 \Delta X_2 + \omega_{21} \Delta X_2 \Delta X_1 + \dots$$

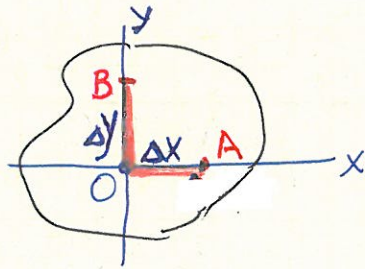


$\Rightarrow \omega_{ij}$ term does not affect length changes

ω_{ij} describe rotations

closer look at ω_{ij} $= \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right)$

Consider a deformable particle



Planar motion of XY cross-section

How do we define angle ω of rotation of the particle?

Not obvious: it's different for different lines

Reasonable definition: take ω for two perpendicular lines (say, X, Y) and average ω 's for them

$$u_y(0) + \omega_{OA} \cdot \Delta x = u_y(A) \Rightarrow \omega_{OA} = \frac{\partial u_y}{\partial x}$$

$$u_x(0) - \omega_{OB} \cdot \Delta y = u_x(B) \Rightarrow \omega_{OB} = -\frac{\partial u_x}{\partial y}$$

↑
clockwise

$$\text{Average} : \frac{1}{2} (\omega_{OA} + \omega_{OB}) = \frac{1}{2} \left(\frac{\partial u_y}{\partial x} - \frac{\partial u_x}{\partial y} \right) = \omega_{xy}$$

$\therefore \omega_{ij}$ gives a physically reasonable definition of angle of rotation of a deformable particle

Comment : the key assumption was :

$$\frac{\partial u_i}{\partial x_j} (= \epsilon_{ij} + \omega_{ij}) \text{ are } \underline{\text{all small}}$$

(small geometry changes)

This means : both - strains ϵ_{ij}
- rotations ω_{ij} are small

excludes : flexible members (membranes, etc)
(strains may be small, but rotations are not)
Such as thin steel sheets

Thus

$$\overline{\Delta S}^2 - \Delta S^2 = 2 \epsilon_{ij} \underbrace{\Delta X_i \Delta X_j}_{\substack{\text{identifies the} \\ \text{element } \Delta X}}$$

Transform this formula, to get relative elongation $\frac{\overline{\Delta S} - \Delta S}{\Delta S}$

$$\overline{\Delta S}^2 - \Delta S^2 = (\overline{\Delta S} - \Delta S) (\overline{\Delta S} + \Delta S)$$

$$\hookrightarrow 2\Delta S + (\overline{\Delta S} - \Delta S)$$

Neglecting the term of second of smallness $(\overline{\Delta S} - \Delta S)^2$:

$$\approx 2\Delta S (\overline{\Delta S} - \Delta S)$$

$$\Rightarrow \frac{\overline{\Delta S} - \Delta S}{\Delta S} = \frac{\epsilon_{ij} \Delta X_i \Delta X_j}{\Delta S^2}$$

represent $\Delta X = \Delta S \underline{n}$ ← orientation info.

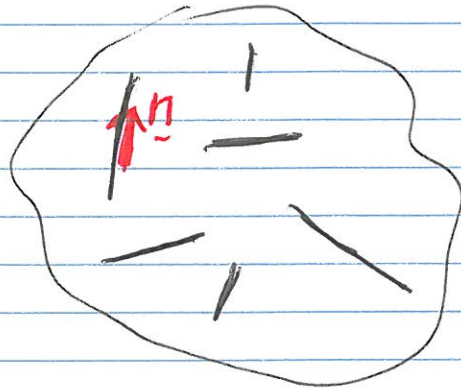
$$\Rightarrow \frac{\overline{\Delta S} - \Delta S}{\Delta S} = \epsilon_{ij} n_i n_j \quad \left. \begin{array}{l} \text{directional info} \\ \text{- basic formula} \\ \text{of strain analysis} \end{array} \right\}$$

Note: independent of choice of sense:

$$\underline{n} \rightarrow -\underline{n}$$



Application : Composite material, with fibers



deformed,
strain tensor ϵ_{ij}

Find : relative elongations (strains)
in fibers of different orientations

— use basic formula —

Diagonal components of strain : for example ϵ_{11}

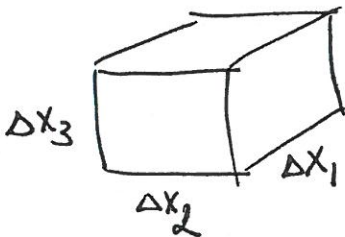
Choose mat'l element along X_1 axis : $n_1 = 1, n_2 = n_3 = 0$

Its relative elongation :

$$\frac{\bar{\Delta S} - \Delta S}{\Delta S} = \epsilon_{ij} n_i n_j = \epsilon_{11}$$

$$\Rightarrow \text{its new length } \bar{\Delta S} = (1 + \epsilon_{11}) \Delta S$$

Sum of the diagonal elements ϵ_{ii} ($= \epsilon_{11} + \epsilon_{22} + \epsilon_{33}$)



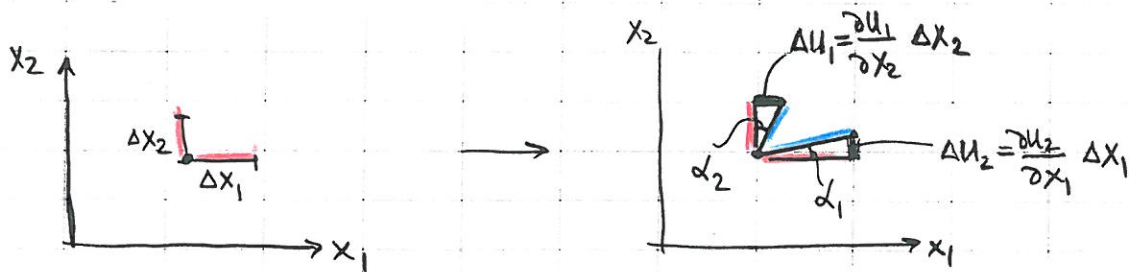
$$\Delta V = \Delta X_1 \Delta X_2 \Delta X_3$$

$$\bar{\Delta V} = \Delta X_1 (1 + \epsilon_{11}) \Delta X_2 (1 + \epsilon_{22}) \Delta X_3 (1 + \epsilon_{33})$$

$$= \Delta V (1 + \epsilon_{11} + \epsilon_{22} + \epsilon_{33} + \text{higher order terms})$$

$$\Rightarrow \epsilon_{ii} = \frac{\bar{\Delta V} - \Delta V}{\Delta V} \quad \text{— relative volume change (dilatation)}$$

Off-Diagonal components of ϵ_{ij} . For example ϵ_{12}



$$\tan \alpha_1 = \frac{\frac{\partial u_2}{\partial x_1} \Delta x_1}{\Delta x_1} = \frac{\partial u_2}{\partial x_1} ; \quad \tan \alpha_2 = \frac{\partial u_1}{\partial x_2}$$

$$\therefore 2\epsilon_{12} = \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} = \tan \alpha_1 + \tan \alpha_2 \approx \alpha_1 + \alpha_2 \quad \text{for small geom. changes}$$

$\therefore 2\epsilon_{12}$ = distortion of the (originally 90°) angle between material lines along x_1 & x_2 directions

Analogously, ϵ_{23} , ϵ_{31} .

Principal Form of Strain

$$\epsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) - \text{symmetric tensor} \quad \Rightarrow \quad 3 \text{ real roots in eigenv. problem}$$

$$\underline{\epsilon} = \epsilon_I \underline{e}_I \underline{e}_I + \epsilon_{II} \underline{e}_{II} \underline{e}_{II} + \epsilon_{III} \underline{e}_{III} \underline{e}_{III}$$

Any deformation: equivalent to elongations & contractions in 3 principal directions; angles do not change between them

Summary on strains

$$\epsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) = \epsilon_{ji} \quad \text{symmetric tensor}$$


$\epsilon_{12} = \epsilon_{21}$, etc

Diagonals:



$$\epsilon_{11} = \frac{\partial u_1}{\partial x_1} \quad - \text{(relative) elongation in } x_1 \text{-direction}$$

off-diagonals:



$$\epsilon_{12} = \frac{1}{2} \left(\frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \right) = \frac{1}{2} \text{ distortion of (originally } 90^\circ \text{ angle } x_1 \wedge x_2$$

$$\epsilon_{ij} = \begin{pmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{21} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{31} & \epsilon_{32} & \epsilon_{33} \end{pmatrix}$$

do not have to distinguish between ϵ_{12} , ϵ_{21}

Symmetric tensor \Rightarrow 3 real eigenvalues & 3 eigenvectors
(princ. axes of $\underline{\epsilon}$)
 \Rightarrow has principal representation:

$$\underline{\epsilon} = \epsilon_I \underline{e}_I \underline{e}_I + \epsilon_{II} \underline{e}_{II} \underline{e}_{II} + \epsilon_{III} \underline{e}_{III} \underline{e}_{III}$$

\Downarrow

Any deformation is equivalent to elongations or contractions
in 3 orthogonal princ. directions