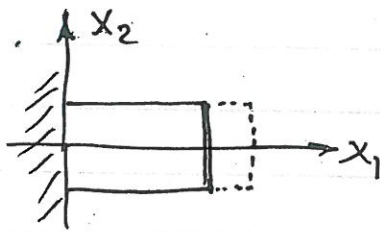


strains - 2

Example : uniaxial strain



↙ extent of straining

$$x_1 \rightarrow \bar{x}_1 = x_1 + kx_1$$

$$\bar{x}_2 = x_2$$

$$\bar{x}_3 = x_3$$

u_1

Displacements :

$$u_1 = kx_1$$

$$u_2 = u_3 = 0$$

Displac. gradient tensor:

$$D_{ij} = \frac{\partial u_i}{\partial x_j} = \begin{vmatrix} k & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{vmatrix}$$

- symmetric (no ω part)

$$= \epsilon_{ij}$$

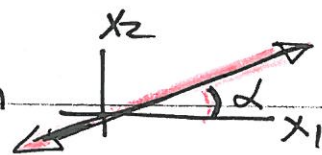
$$\epsilon_{11} = k, \text{ other } \epsilon_{ij} = 0$$

In dyadic form:

$$\underline{\epsilon} = k \underline{e}_1 \underline{e}_1$$

Using strain tensor:

find relative elongation in direction



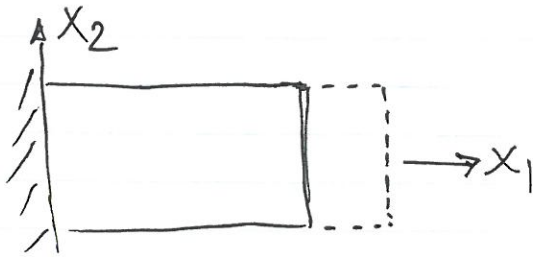
$$\text{Unit vector : } \underline{n} = \frac{\cos \alpha}{n_1} \underline{e}_1 + \frac{\sin \alpha}{n_2} \underline{e}_2$$

$$\frac{\Delta \bar{S} - \Delta S}{\Delta S} = \epsilon_{ij} n_i n_j = \epsilon_{11} n_1^2 = k \cos^2 \alpha$$

Changes from k to 0 as α changes from 0 to 90°

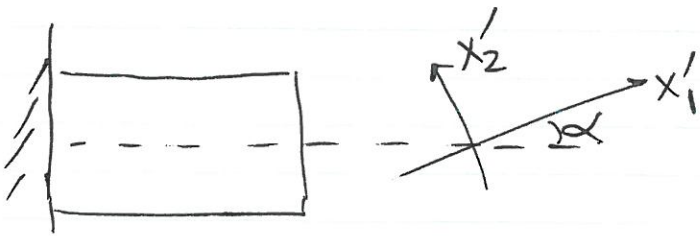
[Note : \cos^2 not \cos !]

Uniaxial straining - cont'd



No angle change between x_1, x_2 directions : $\epsilon_{12} = 0$
 (intuitively obvious)

But what about angle between x'_1, x'_2 ?



$$\underline{\epsilon} = k \underline{e}_1 \underline{e}_1$$

$$\underline{e}_1 = \cos \alpha \underline{e}'_1 - \sin \alpha \underline{e}'_2$$

$$\underline{\epsilon} = k (\cos \alpha \underline{e}'_1 - \sin \alpha \underline{e}'_2) (\cos \alpha \underline{e}'_1 - \sin \alpha \underline{e}'_2)$$

$$= \underbrace{k \cos^2 \alpha}_{\text{relative elong in } x'_1 \text{-dir.}} \underline{e}'_1 \underline{e}'_1 + \underbrace{k \sin^2 \alpha}_{\text{in } x'_2 \text{-dir.}} \underline{e}'_2 \underline{e}'_2 - \underbrace{k \sin \alpha \cos \alpha}_{\epsilon_{12}'} (\underline{e}'_1 \underline{e}'_2 + \underline{e}'_2 \underline{e}'_1)$$

recover:

relative elong
in x'_1 -dir.

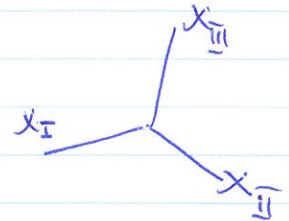
in x'_2 -dir.

ϵ_{12}'
 \uparrow $\frac{1}{2}$ angle distortion

Basic formula of strain analysis

$$\frac{\bar{\Delta S} - \Delta S}{\Delta S} = \epsilon_{ij} n_i n_j$$

in principal axes of strain

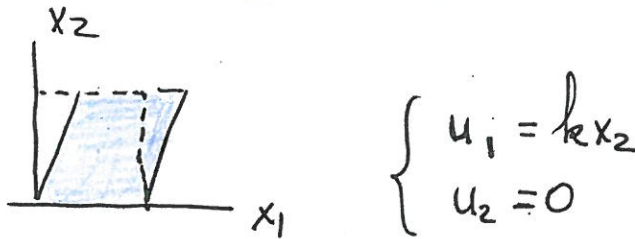


$$\underline{\underline{\epsilon}} = \epsilon_I \underline{e}_I \underline{e}_I + \epsilon_{II} \underline{e}_{II} \underline{e}_{II} + \epsilon_{III} \underline{e}_{III} \underline{e}_{III} \quad \text{no off-diagonals}$$

$$\underline{n} = n_I \underline{e}_I + n_{II} \underline{e}_{II} + n_{III} \underline{e}_{III}$$

$$\Rightarrow \frac{\bar{\Delta S} - \Delta S}{\Delta S} = \epsilon_I n_I^2 + \epsilon_{II} n_{II}^2 + \epsilon_{III} n_{III}^2$$

Example : Shear strain



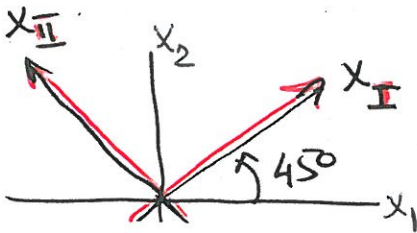
$$D_{ij} = \frac{\partial u_i}{\partial x_j} = \begin{vmatrix} 0 & k \\ 0 & 0 \end{vmatrix} \text{ - not symmetric}$$

$$\underline{\underline{\epsilon}}_{ij} = \begin{vmatrix} 0 & k/2 \\ k/2 & 0 \end{vmatrix} \quad \underline{\underline{\epsilon}} = \frac{k}{2} (\underline{e}_1 \underline{e}_2 + \underline{e}_2 \underline{e}_1) \quad \text{(no volume change } \epsilon_{ii} = 0)$$

Eigenvalue problem:

$$\det \begin{vmatrix} -\lambda & k/2 \\ k/2 & -\lambda \end{vmatrix} = 0 \Rightarrow \lambda_{1,2} = \pm k/2$$

$$\text{Eigenvectors: } \lambda_1 = \frac{k}{2} \quad \left. \begin{aligned} -\frac{k}{2}x_1 + \frac{k}{2}x_2 &= 0 \\ \frac{k}{2}x_1 - \frac{k}{2}x_2 &= 0 \end{aligned} \right\}$$



$$\Rightarrow x_1 = x_2$$

$$\lambda_2 = -k/2 \Rightarrow x_1 = -x_2$$

Principal axes representation $\underline{\underline{\epsilon}} = \frac{k}{2} (\underline{e}_I \underline{e}_I - \underline{e}_{II} \underline{e}_{II})$ - shear strain is equiv. to stretch & contract. at 45° directions

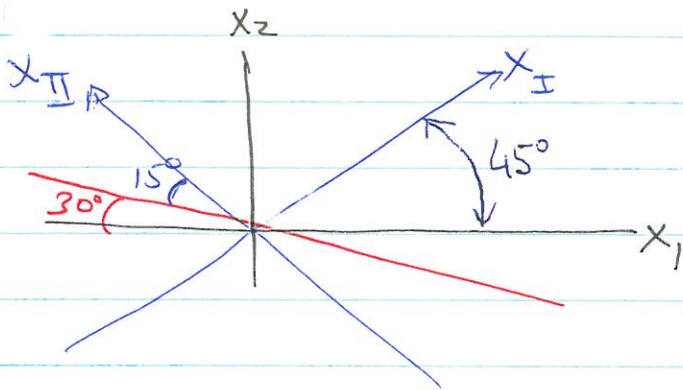
relative elongation in direction \rightarrow

Working in x_1, x_2 axes:

$$\underbrace{-\frac{\sqrt{3}}{2}}_{n_1} \underline{e}_1 + \underbrace{\frac{1}{2}}_{n_2} \underline{e}_2 = \underline{n} \text{ - unit vector of this direction}$$

$$\frac{\Delta \bar{S} - \Delta S}{\Delta S} = \epsilon_{ij} n_i n_j = \frac{k}{2} (n_1 n_2 + n_2 n_1) = -\frac{\sqrt{3}}{4} k \quad \leftarrow \text{contraction}$$

→ same calculation in the principal axes



$$\epsilon_I = \frac{k}{2} \quad \epsilon_{II} = -\frac{k}{2}$$

$$\underline{\epsilon} = \frac{k}{2} (\underline{e}_I \underline{e}_I - \underline{e}_{II} \underline{e}_{II})$$

Unit vector of the same direction in the princ. coord. system:

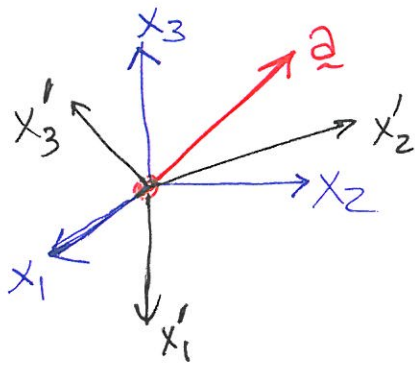
$$\underline{n} = \underbrace{-\sin 15^\circ}_{n_I} \underline{e}_I + \underbrace{\cos 15^\circ}_{n_{II}} \underline{e}_{II}$$

$$\frac{\Delta \bar{S} - \Delta S}{\Delta S} = \epsilon_I n_I^2 + \epsilon_{II} n_{II}^2 =$$

$$= \frac{k}{2} \left(\underbrace{\sin^2 15^\circ - \cos^2 15^\circ}_{-\cos 30^\circ} \right) = -\frac{\sqrt{3}}{4} k \quad \text{-same}$$

tensor invariants

Components of vectors and tensors change when coord system is rotated



Certain combinations of components do not change
invariants

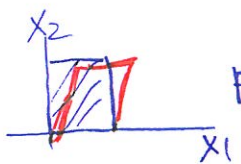
Vector: its length is invariant:

$$a_1^2 + a_2^2 + a_3^2 = a_1'^2 + a_2'^2 + a_3'^2 = |a|^2$$

Tensor: has 3 invariants

One of them: sum of diagonal elements
(called "first", or "linear" invariant)

Strain: $\epsilon_{ii} = \text{relative volume change} = \epsilon'_{ii}$



pure shear

$$\epsilon_{ij} = \begin{pmatrix} 0 & k/2 \\ k/2 & 0 \end{pmatrix} \text{ in } x_1 x_2 \text{ system}$$

$$= \begin{pmatrix} k/2 & 0 \\ 0 & -k/2 \end{pmatrix} \text{ in the princ. system}$$

$\swarrow \searrow$
45°

Sum $\epsilon_{ii} = 0$ in both

Example: $\underline{\underline{\epsilon}} = d \cdot \underline{\underline{I}}$ ($\epsilon_{ij} = d \cdot \delta_{ij}$)

Dilatancy: $\epsilon_{kk} = 3d$

For any material line, with \underline{n} :

$$\frac{\Delta \bar{s} - \Delta s}{\Delta s} = \epsilon_{ij} n_i n_j = d (n_1^2 + n_2^2 + n_3^2) = d$$

same for all orientations

uniformly heated mat'l (if $d < 0$ - uniformly cooled)

Unit tensor $\underline{\underline{I}}$ has the same δ_{ij} matrix
in any coord system - no angle distortions
between any directions

Maximal Relative Elongation

(important for fracture)

$$\max \frac{\Delta \bar{S} - \Delta S}{\Delta S}$$
 (with respect to line orientation)

- equals to...?
- on which orientation?

In the principal axes

$$\underline{\epsilon} = \epsilon_I \underline{e}_I \underline{e}_I + \epsilon_{II} \underline{e}_{II} \underline{e}_{II} + \epsilon_{III} \underline{e}_{III} \underline{e}_{III}$$

↑
 the largest
 (convention, for certainty)

Line Orientation $\underline{n} = n_I \underline{e}_I + n_{II} \underline{e}_{II} + n_{III} \underline{e}_{III}$

$$\begin{aligned} \frac{\Delta \bar{S} - \Delta S}{\Delta S} &= \epsilon_I n_I^2 + \epsilon_{II} n_{II}^2 + \epsilon_{III} n_{III}^2 = \\ &= \epsilon_I \underbrace{(n_I^2 + n_{II}^2 + n_{III}^2)}_1 - \underbrace{[(\epsilon_I - \epsilon_{II}) n_{II}^2 + (\epsilon_I - \epsilon_{III}) n_{III}^2]}_{\geq 0} \end{aligned}$$

- ⇒
- max for $n_{II} = n_{III} = 0$ — along X_I direction.
 - equals ϵ_I

Minimal elongation - similarly, is equal to ϵ_{III} and is along the III-dir.

Note : min / max - with the account of signs
(negative for contraction)

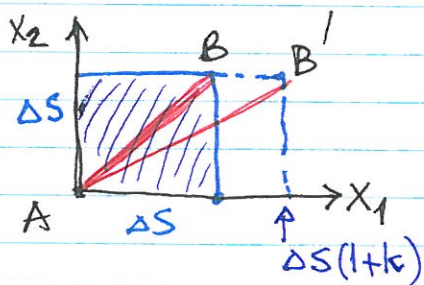
Note on Smallness of Strains & Rotations

We assume all $|\frac{\partial u_i}{\partial x_j}| \ll 1$ (not only ϵ_{ij} , but ω_{ij} as well)

Only then $(\epsilon_{ij}, \omega_{ij})$ have the meanings we identified

Example

uniaxial elongation



$$u_1 = kx_1 \quad u_2 = u_3 = 0 \quad \Rightarrow \quad \epsilon_{11} = k$$

Elongation of diagonal (Pythagorean theorem):

$$AB' - AB = [\sqrt{(1+k)^2 + 1} - \sqrt{2}] \Delta s$$

$$\text{Relative elong: } \frac{[\sqrt{(1+k)^2 + 1} - \sqrt{2}] \Delta s}{(\Delta s \cdot \sqrt{2})}$$

From strains:

$$\epsilon_{11} = k, \text{ other } \epsilon_{ij} = 0$$

$$\text{relative elong: } \epsilon_{ij} n_i n_j = \epsilon_{11} \left(\frac{1}{\sqrt{2}}\right)^2 = \frac{1}{2} k \quad \text{- differs!}$$

But, if k is small - the same

Indeed: $\sqrt{(1+k)^2 + 1} \approx \sqrt{2 + 2k} = \sqrt{2} \sqrt{1+k}$ (small k)

recall: $\sqrt{1+k} \approx 1 + \frac{1}{2}k$ (Taylor series)

\Rightarrow Pythagorean theorem yields $\frac{1}{2}k$

What if:

ϵ_{ij} and ω_{ij} are not small?
then $\frac{\partial u_i}{\partial x_j}$ quantities do not mean anything

Example

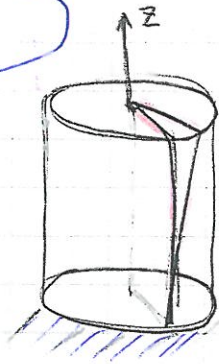
Torsion of a circular cylinder - Strains

Notation:

X_1, X_2, X_3



x, y, z



Kinematics of torsion:

- Each "slice" rotates in its own plane
- The angle of rotation of a given slice is proportional to its distance from the base

$$\theta = \alpha z$$

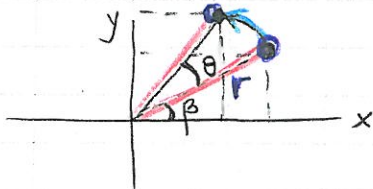
(twist per unit length)

Strains?

Displacements:

(1) $u_z = 0$

(2) To derive u_x, u_y , consider rotation of a given slice



$$u_x = r \cos(\beta + \theta) - r \cos \beta =$$
$$= r [\cos \beta \cos \theta - \sin \beta \sin \theta - \cos \beta]$$

for small θ : $\cos \theta \approx 1$
 $\sin \theta \approx \theta$

$$u_x = r [\cos \beta - \theta \sin \beta - \cos \beta] =$$
$$= -\theta \underbrace{r \sin \beta}_y = -\theta y = \underline{\underline{-\alpha y z}}$$

$$u_y = r [\sin(\beta + \theta) - \sin \beta] = \theta x = \underline{\underline{\alpha x z}}$$

$$D_{ij} = \frac{\partial u_i}{\partial x_j} = \begin{vmatrix} 0 & -\alpha z & -\alpha y \\ \alpha z & 0 & \alpha x \\ 0 & 0 & 0 \end{vmatrix}$$

$$\epsilon_{ij} = \begin{vmatrix} 0 & 0 & -\frac{\alpha}{2} y \\ 0 & 0 & \frac{\alpha}{2} x \\ -\frac{\alpha}{2} y & \frac{\alpha}{2} x & 0 \end{vmatrix}$$

- zero strains at centerline $x=y=0$
- non-uniform strains!
depend on x, y (not on z)

- no volume change
- no angle distortions in plane: $\epsilon_{xy}=0$
- there are distortions of angles between the z - and -inplane directions

Eigenvalue problem: at some point (x_0, y_0) have to specify non uniform ϵ

$$\det \begin{vmatrix} -\lambda & 0 & -\frac{\alpha}{2} y_0 \\ 0 & -\lambda & \frac{\alpha}{2} x_0 \\ -\frac{\alpha}{2} y_0 & \frac{\alpha}{2} x_0 & -\lambda \end{vmatrix} = -\lambda^3 + \frac{\alpha^2}{4} \lambda y_0^2 + \frac{\alpha^2}{4} \lambda x_0^2 = 0$$

$$\lambda_{1,2} = \pm \frac{\alpha}{2} \sqrt{x_0^2 + y_0^2} = \pm \frac{\alpha}{2} r_0 \quad \lambda_3 = 0$$

proportional to distance from the axis

this state of strain is shear:

two equal princ. strains of opposite sign

$$\epsilon_{IJ} = \begin{pmatrix} \frac{\alpha}{2} r_0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & -\frac{\alpha}{2} r_0 \end{pmatrix}$$